Total No. of Questions : 12]		SEAT No. :
P1697	[4859]-37	[Total No. of Pages : 4
	B.E. (Mechanical)	

c - DESIGN OF PUMPS BLOWERS AND COMPRESSORS (Elective - I) (2008 Course) (Semester - I)

Time: 3 Hours] [Max. Marks: 100

Instructions to the candidates:

- 1) Answer any three questions from each section.
- 2) Answers to the two sections should be written in separate books.
- 3) Neat diagrams must be drawn wherever necessary.
- 4) Figures to the right indicate full marks.
- 5) Use of Logarithmic tables slide rule, Mollier charts, and electronic pocket calculator and steam tables are allowed.
- 6) Assume suitable data, if necessary.

SECTION - I

Q1) a) Explain the following terms:

[8]

- i) Flow Machines
- ii) Turbines
- iii) Pumps
- iv) Compressible Flow Machines
- b) A turbo blower develops 750 mm W.G. at a speed of 1480 rpm and a flow rate of 38m³/s. It is desired to build a small model which develops the same head at a higher speed (2490 rpm) and low discharge. Determine the specific speed and the flow rate through the model. [8]

OR

- **Q2)** a) Explain the performance characteristics of pumps, compressors, fans and blowers. [10]
 - b) Write equations of energy transfer between fluid and rotor. [6]

- Q3) a) The impeller of a centrifugal pump has 1.4m outside diameter. It is used to lift 1800 liters of water per second against a head of 10 m. Its Vanes make an angle of 45° with the direction of motion at outlet and runs at 400 rpm. If the radial velocity of flow at outlet is 3.5 m/s, find the manometric efficiency. Also find the power required if the overall efficiency is 82%.
 - b) Explain various efficiencies of centrifugal pump.

[8]

OR

- **Q4)** a) Explain various types of characteristic curves usually prepared for centrifugal pumps. [8]
 - b) What is NPSH? Derive the expression of the same. Find the height from the water surface at which a centrifugal pump may be installed in the following case to avoid cavitation: Atmospheric pressure = 1.01 bar; vapour pressure = 0.022 bar; losses in suction pipe = 1.42 m; effective head of pump = 49m; and cavitation factor = 0.115. [8]
- **Q5)** a) Explain the following terms:

[8]

- i) Static Suction Head
- ii) Static Discharge Head
- iii) Total Static Head
- b) Explain the design procedure of centrifugal pump.

[10]

OR

- **Q6)** a) Explain various forms of corrosion occurred in hydraulic machines. [8]
 - b) A centrifugal pump running at 1450 rpm has the characteristic as given below: [10]

Discharge (Lit/sec)	11.3	16.9	22.6	28.3	34	39.6	45.2
Head (m)	25.8	25	24.1	23.2	21.4	18.9	15.8
Efficiency %	65	70	73	74	72	69	62

Draw the operating characteristic of the pump and determine its specific speed. The pump lifts water against a static head of 12 m through a long pipeline in which the loss of head in meters, due to friction is given by the expression, hf = 0.012 Q2, where Q is the discharge in liters/sec. The minor losses in the pipe may be neglected. Determine the power required to drive the pump.

SECTION - II

Q7) a) Discuss various applications of fan.

[8]

b) Explain functions of an airfoil and discuss the characteristics curves of airfoils. [8]

OR

- **Q8)** a) How does dust erosion of centrifugal pump impeller occurs? What is its effect on the performance? [8]
 - b) Prove the following relations for an axial fan stage with UGV and DGVS:→

$$(\Delta p)$$
 st = $2 \rho u^2 (\phi \tan \beta_2 - 1)$, $\psi = 4(\phi \tan \beta_2 - 1)$ and R = 1 [8]

- **Q9)** a) What are the main causes of noise generation? What are the methods of reducing fan noises? [8]
 - b) What is surging? What are its effects? What is stalling? How it is developed? [8]

OR

- Q10)a) Explain briefly what is the purpose of inlet guide vanes and inducer blades. Why is the radial tipped impeller mostly used in centrifugal compressor stages?[8]
 - b) Stage design considerations and empirical relations used to determine various fan design parameters. [8]
- Q11)a) What is the work done factor for an axial compressor stage? How does it vary with the number of stages?[8]
 - b) Prove the following relation for isentropic flow in a radial tipped impeller.

$$\psi$$
=1; P_{rw} =1+ $\left(\frac{U_2^2}{Cp.T01}\right)$: (y/y-1); with usual notations. [10]

OR

Q12)a) Derive the following relations for an axial compressor stage with constant axial velocity. [12]

i)
$$(\tan \alpha_1 + \tan \beta_1) = (\tan \alpha_2 + \tan \beta_2) = (U/Vx)$$

ii)
$$\psi = \phi (\tan \beta_1 - \tan \beta_2)$$

iii)
$$\frac{(\Delta p)st}{\rho u^2} = \phi (\tan \alpha_2 - \tan \alpha_1)$$

iv)
$$(n_{st}) = \left(\frac{(\Delta p)st}{\Omega \rho UV_x (\tan \alpha_2 - \tan \alpha_1)}\right)$$
; with usual notations.

b) What is "slipfactor"? What is its effect on the flow and the pressure ratio in the stage? [6]

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