

**B.E. (Mechanical)****FINITE ELEMENT METHOD****(2008 Course) (Semester - II) (402049 - B) (Elective - III)***Time : 3 Hours]**[Max. Marks : 100**Instructions to the candidates:*

- 1) *Answer to the two sections should be written in separate answer books.*
- 2) *Figures to the right indicates full marks.*
- 3) *Use of calculator is allowed.*
- 4) *Assume suitable data if necessary.*
- 5) *Additional data sheet is attached for the reference.*

**SECTION - I****UNIT - I**

- Q1)** a) Write a short note on numbering nodes for band width minimization. [6]  
 b) List and briefly describe the general steps of the finite element method. [4]  
 c) What is meant by plane stress and plane strain condition? State the relationship between stress-strain for plane stress and plane strain condition in 2D elasticity. [6]

**OR**

- Q2)** a) With the help of neat sketch explain axisymmetric formulations in elasticity? State stress-strain relation for axisymmetric problems. [6]  
 b) State and explain the principle of minimum potential energy (PMPE). [4]  
 c) Explain the term skyline storage technique. [6]

**UNIT - II**

- Q3)** a) Derive equation for load vector due to body force and traction force for two noded (linear) bar element. [6]  
 b) Find slopes and deflections of beam subjected to uniform distributed load as shown in Figure 1. Take modulus of elasticity,  $E = 2 \times 10^{11} \text{ N/m}^2$  and  $I = 5 \times 10^{-6} \text{ m}^4$  [12]

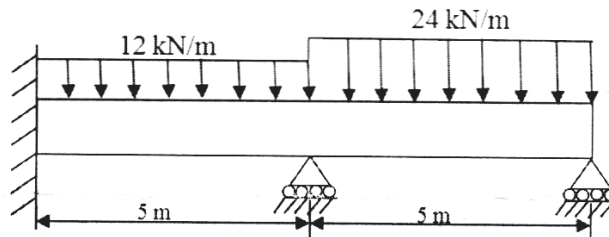


Figure 1

**OR**

- Q4) a)** Derive the expression for load vector for beam element due to distributed load P. [6]
- b)** For the bar as shown in Figure 2, determine the nodal displacements and element stresses. Take modulus of elasticity  $E = 2 \times 10^{11} \text{ N/m}^2$ . Take maximum displacement at free end as 3.5 mm. Use elimination approach for boundary conditions. [12]

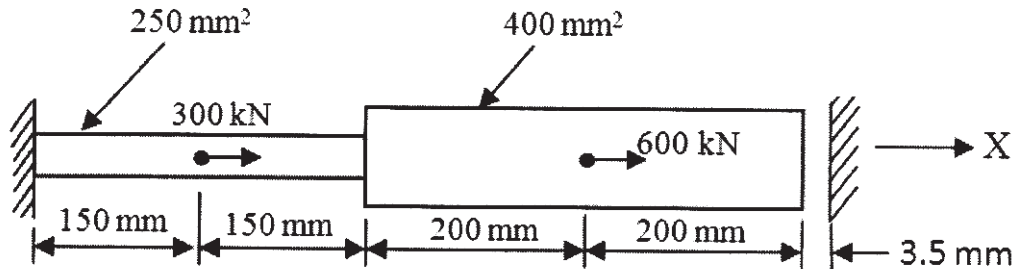


Figure 2

### UNIT - III

- Q5) a)** Derive the elements of Jacobian matrix of transformation for 2D plane stress condition using four noded quadrilateral element. [8]
- b)** For four node quadrilateral element, find the x and y coordinates of point P whose location in parent element are given by  $\xi = 0.5$  and  $\eta = 0.5$ . Also find u, v displacements of point P in X and Y directions respectively if displacement vector is  $\{q\} = [0 \ 0 \ 0.20 \ 0 \ 0.15 \ 0.10 \ 0 \ 0.05]^T$ . [8]

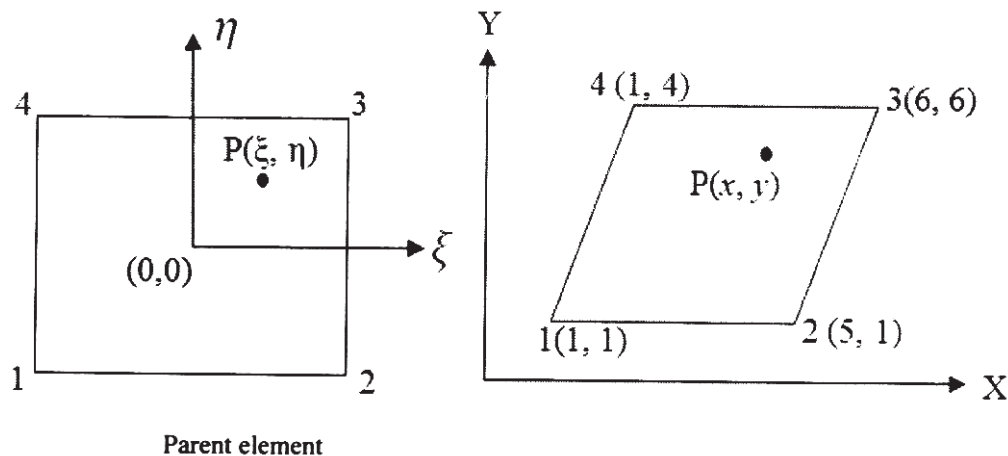


Figure 3

OR

- Q6)** a) Write a note on substructuring. [4]  
 b) Explain one dimensional Gauss Quadrature rule for numerical integration. [6]  
 c) Solve following integration using two point Gauss Quadrature method. [6]

$$I = \int_{-1}^1 \int_{-1}^1 (2\xi^2\eta + \eta^2\xi - 3\xi\eta + 7) d\xi d\eta$$

## SECTION - II

### UNIT - IV

- Q7)** a) Derive elemental stiffness matrix formulations for one dimensional steady state heat conduction problems. [8]  
 b) Consider a brick wall of thickness 0.3 m,  $k = 0.7 \text{ W/m}^\circ\text{K}$  and area  $A = 1 \text{ m}^2$  perpendicular to the direction of heat flow. The inner surface is at  $28^\circ\text{C}$  and the outer surface is exposed to cold air at  $-15^\circ\text{C}$ . the heat transfer coefficient associated with the outside surface is  $40 \text{ W/m}^2 \text{ }^\circ\text{K}$ . Determine the steady state temperature distribution within the wall. Use two elements and use elimination approach for boundary conditions. [10]

OR

- Q8)** a) Derive elemental stiffness matrix (conduction + convection) formulations for 1D steady state heat transfer problems. [8]  
 b) A metallic fin, with thermal conductivity  $70 \text{ W/m}^\circ\text{K}$ , 1 cm radius and 5 cm long extends from a plane wall whose temperature is  $140^\circ\text{C}$ . Determine the temperature distribution along the fin if heat is transferred to ambient air at  $20^\circ\text{C}$  with heat transfer coefficient of  $5 \text{ W/m}^2 \text{ }^\circ\text{K}$ . Take two elements along the fin. Assume that the tip of fin is insulated. [10]

### UNIT - V

- Q9)** Estimate natural frequencies of axial vibrations of bar shown in figure 4. using both consistent and lumped mass matrices and compare the results. Bar is having uniform cross-section with cross-sectional area  $A = 30 \times 10^{-6} \text{ m}^2$ , length  $L = 1 \text{ m}$ , modulus of elasticity  $E = 2 \times 10^{11} \text{ N/m}^2$  and density  $\rho = 7800 \text{ kg/m}^3$ . Model the bar by using two elements. [16]

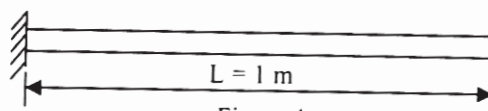


Figure 4

OR

- Q10)a)** Derive consistent mass matrix for truss element and one dimensional bar element. **[8]**
- b) Explain eigen value problem for un-damped free vibration system. **[8]**

### **UNIT - VI**

- Q11)a)** Explain various types of analysis in finite element method. **[8]**
- b) What are the symmetric, and antisymmetric boundary conditions in finite element method? Illustrate with examples. **[8]**

OR

- Q12)a)** Define: Aspect ratio, warp angle, Skews, Jacobian. Explain their significance in finite element method. **[8]**
- b) What are the advantages and limitations of free and mapped meshing in Finite Element Method? Which is most suitable for meshing complex geometries? **[8]**

## DATA SHEET

### Shape Functions:

#### 1 Bar Element

Cartesian coordinates

$$N_1 = 1 - \frac{x}{L}; \quad N_2 = \frac{x}{L};$$

Natural coordinates

$$N_1 = \frac{1-\xi}{2}; \quad N_2 = \frac{1+\xi}{2}$$

#### 2 Beam Element

In Cartesian coordinate

$$N_1 = \frac{1}{L^3}(2x^3 - 3x^2L + L^3);$$

$$N_2 = \frac{1}{L^3}(x^3L - 2x^2L^2 + xL^3);$$

$$N_3 = \frac{1}{L^3}(-2x^3 + 3x^2L);$$

$$N_4 = \frac{1}{L^3}(x^3L - x^2L^2);$$

In Natural coordinate

$$N_1 = \frac{1}{4}(2 - 3\xi + \xi^3);$$

$$N_2 = \frac{1}{4}(1 - \xi - \xi^2 + \xi^3);$$

$$N_3 = \frac{1}{4}(2 + 3\xi - \xi^3);$$

$$N_4 = \frac{1}{4}(-1 - \xi + \xi^2 + \xi^3);$$

### Element Stiffness Matrices:

#### 1. Bar Element

$$k_{bar} = \frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

#### 2. Beam Element

$$k_{beam} = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L & -12 & 6L \\ 6L & 4L^2 & -6L & 2L^2 \\ -12 & -6L & 12 & -6L \\ 6L & 2L^2 & -6L & 4L^2 \end{bmatrix}$$

#### 3 Truss Element

$$k_{truss} = \frac{AE}{L} \begin{bmatrix} C^2 & CS & -C^2 & -CS \\ CS & S^2 & -CS & -S^2 \\ -C^2 & -CS & C^2 & CS \\ -CS & -S^2 & CS & S^2 \end{bmatrix};$$

$C = \cos(\theta)$  and  $S = \sin(\theta)$ ,  $\theta$  is positive in anti-clockwise direction

## Element Mass Matrices:

### 1. Bar Element

(a) Consistent mass matrix:

$$m_{consistent} = \frac{\rho AL}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix};$$

(b) Lumped mass matrix:

$$m_{lumped} = \frac{\rho AL}{2} \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

### 2. Beam Element

(a) Consistent mass matrix:

$$m_{consistent} = \frac{\rho AL}{420} \begin{bmatrix} 156 & 22L & 54 & -13L \\ 22L & 4L^2 & 13L & -3L^2 \\ 54 & 13L & 156 & -22L \\ -13L & -3L^2 & -22L & 4L^2 \end{bmatrix};$$

(b) Lumped mass matrix:

$$m_{lumped} = \frac{\rho AL}{2} \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

## Heat Transfer Matrices:

K matrix for conduction + convection problem for bar element

$$K = \frac{kA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} + \frac{hPL}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}; \text{ when end of fin is insulated}$$

$$K = \frac{kA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} + \frac{hPL}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix} + hA \begin{bmatrix} 0 & 0 \\ 0 & 1 \end{bmatrix}; \text{ when end of fin is not insulated}$$

Where  $A$  = cross-sectional area,  $k$  = Thermal conductivity,  $L$  = length of element,

$h$  = convection coefficient, and  $P$  = perimeter

## Gauss Quadrature:

Table for Gauss points for integration from -1 to +1

Number of Gauss Points	Location $\xi_i$	Associated weights
1	$\xi_1 = 0.0$	2.0
2	$\xi_1, \xi_2 = \pm 0.57735$	1.0
3	$\xi_1, \xi_3 = \pm 0.77459$ $\xi_2 = 0.0$	$5/9 = 0.55556$ $8/9 = 0.88889$
4	$\xi_1, \xi_4 = \pm 0.86113$ $\xi_2, \xi_3 = \pm 0.33998$	0.34785 0.65214

x x x