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[5152]-119

S.E. (Mechanical/Mechanical-SW/Automobile)

EXAMINATION, 2017

STRENGTH OF MATERIALS

(2012 COURSE)

Time : Two Hours

Maximum Marks : 50

N.B. :— (i) Solve Q. No. 1 or Q. No. 2, Q. No. 3 or Q. No. 4,
Q. No. 5 or Q. No. 6 and Q. No. 7 or Q. No. 8.

(ii) All the *four* questions should be solved in one answer book
and attach the extra supplements if required.

(iii) Neat diagrams must be drawn wherever necessary.

(iv) Figures to the right indicate full marks.

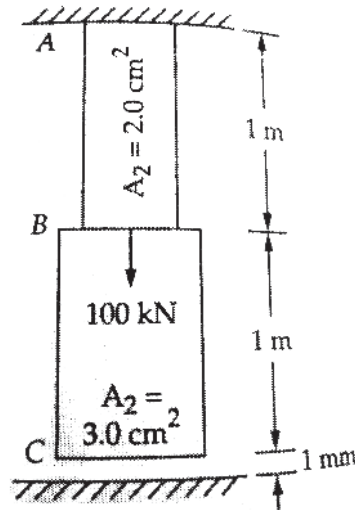
(v) Assume suitable data, if necessary.

1. (A) A bar ABC shown in figure consist of two parts, AB and BC, each part being 1 m long and having cross-sectional areas 2 cm^2 and 3 cm^2 respectively. The bar is suspended from A and there is a rigid horizontal support at 2.001 m from A. A force of 100 kN acting vertically downwards is applied at

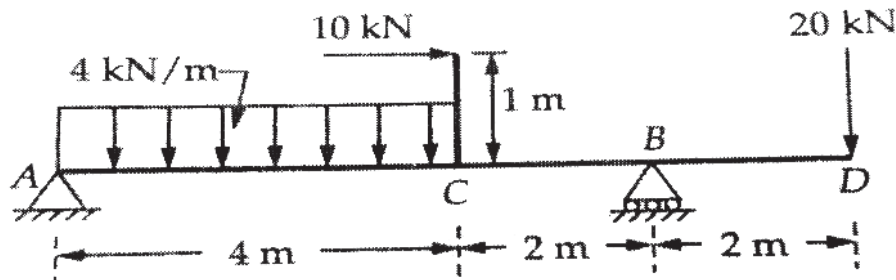
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B. Determine the stresses in parts AB and BC of the bar.

Take $E = 200 \text{ GN/m}^2$. [6]

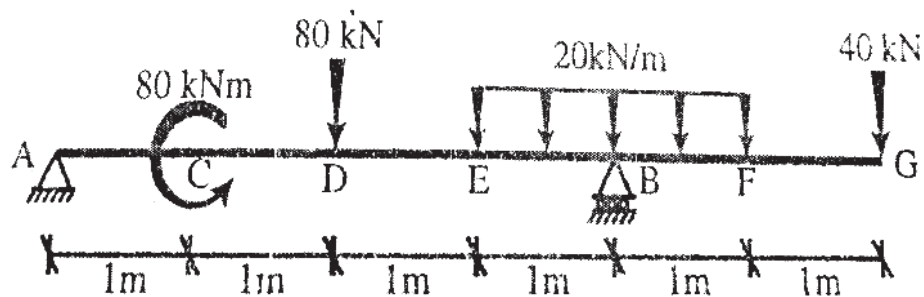


(B) Draw SFD and BMD for the beam shown in the following figure. Also locate the points of contraflexure if any. [6]



Or

2. (A) The beam is supported and loaded as shown in figure. Draw SFD and BMD indicating all important values. [6]



(B) A bar of steel is 40 mm × 40 mm in cross-section and it is 120 mm long. It is subjected to a tensile of 200 kN along the longitudinal axis and tensile loads of 500 kN & 400 kN on the lateral faces.

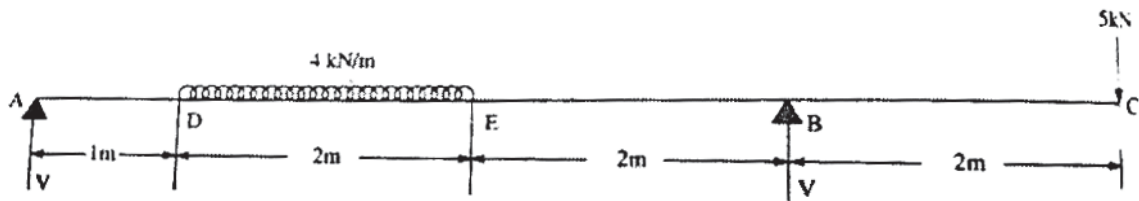
(i) Find change in dimensions of the bar and change in volume.

(ii) Find also what axial longitudinal tensile load acting alone can produce the same longitudinal strain as above case.

Take $E = 2 \times 10^5 \text{ N/mm}^2$, $\mu = 0.3$. [6]

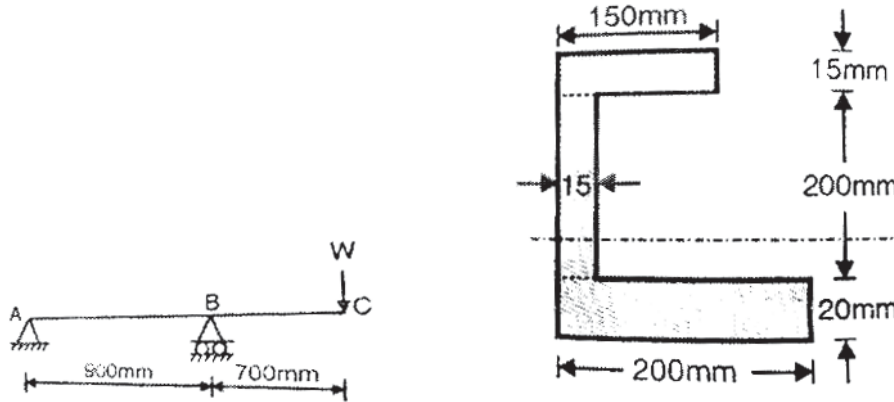
3. (A) For the loaded beam shown in figure. Find the deflection at free end and the maximum deflection between the supports.

Take $E = 200 \text{ kN/mm}^2$ and $I = 9 \times 10^6 \text{ mm}^4$. [6]



(B) A simply supported overhang is loaded with point load as shown in figure. A CI beam of C section with top flange 150 mm × 15 mm, bottom flange 200 mm × 20 mm and web 15 mm × 200 mm. The allowable stresses in tension and com-

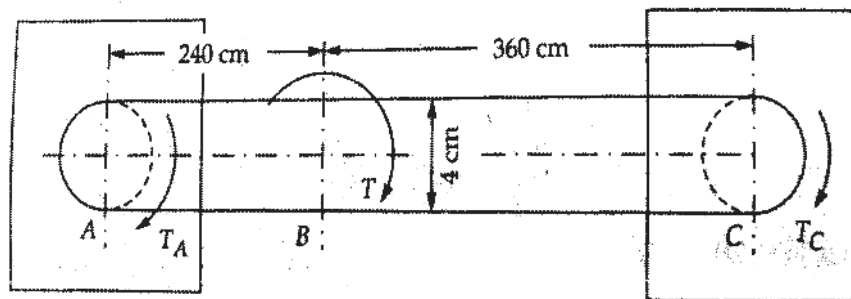
pression are 120 MPa and 90 MPa. Find the safe value of load 'W' on the overhang. [6]



Or

4. (A) A vertical steel rod, 2 m long is fixed at its upper end and a weight sliding freely on the rod falls on the collar fixed firmly at its end. When the weight falls through a height of 15 mm, the maximum stress 60 MPa. What will be the maximum instantaneous stress developed if the same weight had been dropped through height of 25 mm ?
 $E = 210 \text{ GPa}$. [6]
- (B) A symmetrical beam of I-section is 200 mm \times 400 mm in size, the thickness of flange 20 mm and web is 15 mm. The beam carrying UDL of 25 kN/m over entire length of 5 m. Draw the shear stress distribution diagram over depth of section. Also find the ratio of maximum shear stress to average shear stress. [6]

5. (A) Find Euler's critical load for a hollow cylindrical steel column of 40 mm external diameter and 3 mm thick. Take length of column is 3 m and hinged at its both ends. Also determine crippling load by Rankine's formula. Take $\alpha = 1/7500$, $E = 205 \text{ kN/mm}^2$ and $\sigma_c = 335 \text{ N/mm}^2$. For what length of column would critical load by Euler's and Rankine's formula be equal ? [7]
- (B) A 600 cm long solid shaft is fixed at both ends. A torque of 75 kN cm is applied to the shaft at a section of 240 cm from one end. What are the fixing torques set up at the ends of the shaft ? If the diameter of the shaft is 4 cm, calculate the maximum stress developed in the two portions. Also find the angle of twist at the point where the torque is applied. Take $G = 75 \text{ kN/cm}^2$. [6]



Or

6. (A) Determine the ratio of the buckling strengths of a solid steel column to that of a hollow column of same material and having same cross-sectional area. The internal diameter of hollow column is half of its external diameter. Both the columns are of the same length and are pinned at both ends. [7]

- (B) A hollow shaft of diameter ratio $3/5$ is required to transmit 482 KW at 125 rpm. The shearing stress in the shaft must not to exceed 65 N/mm^2 and the twist in a length of 2 m not to exceed 1 degree. Calculate minimum external diameter of shaft which would satisfy these conditions. Take $G = 8 \times 10^4 \text{ N/mm}^2$. [6]

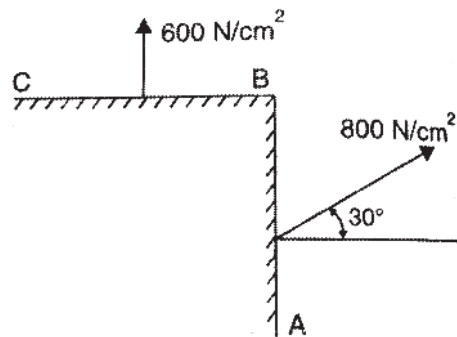
7. (A) A bolt is subjected to an axial pull of 12 kN and a transverse shear force of 5 kN. Determine the diameter of the bolt required based on : [6]

- (i) Maximum principal stress theory
- (ii) Maximum distortion energy theory.

Take elastic limit in simple tension is equal to 270 MPa and Poisson's ratio = 0.3. Adopt F.O.S. = 3.

- (B) The intensity of resultant stress on a plane AB as shown in figure at a point in a material under stress in 800 N/cm^2 and it is inclined at 30° on the normal to that plane. The normal component of stress on another plane BC at right angles to plane AB is 600 N/cm^2 . Determine : [7]

- (i) the resultant stress on the plane BC,
- (ii) the principal stresses and their directions and
- (iii) the maximum shear stress.



Or

8. (A) A solid circular shaft is subjected to a bending moment of 8 kNm and a torque of 12 kNm. In a uniaxial test the shaft material gave the following results : Modulus of elasticity = 200 GN/m², Stress at yield point = 300 N/mm², Poisson's ratio = 0.3, Factor of safety = 3. Estimate the least diameter of the shaft using : [6]

(i) Maximum shear stress theory

(ii) Shear strain energy theory.

- (B) At a point in a strained material, two-dimensional state of stress is as shown in figure. Determine graphically : [7]

(i) Principal stresses

(ii) Principal planes

(iii) Maximum shear stress

(iv) Normal and shear stress component on planes whose normals are at 35° and 125° with x -axis.

