

Total No. of Questions : 12]

SEAT No. :

**P2389**

[Total No. of Pages :5

**[5153]- 12**  
**T.E. (Mech.)**  
**HEAT TRANSFER**  
**(2008 Pattern) (Semester - II)**

*Time : 3 Hours]*

*[Max. Marks :100*

*Instructions to the candidates:*

- 1) Answers to the two sections should be written in separate answer books.*
- 2) Answer any three questions from each section.*
- 3) Neat diagrams must be drawn wherever necessary.*
- 4) Figures to the right side indicate full marks.*
- 5) Use of Calculator is allowed.*
- 6) Assume Suitable data if necessary.*

**SECTION - I**

- Q1)** a) How does the thermal conductivity of insulating material vary with temperature? Explain with the help of suitable sketch. **[4]**
- b) Define thermal conductivity and thermal diffusivity. State their units. **[4]**
- c) A refrigerator stands in a room. Where air temperature is  $21^{\circ}\text{C}$ . The surface temperature on the outside of the refrigerator is  $16^{\circ}\text{C}$ . The sides are 30 mm Thick and has an equivalent thermal conductivity of  $0.10 \text{ W/m} \cdot \text{K}$ . The heat transfer coefficient on the outside is  $10 \text{ W/m}^2 \cdot \text{K}$ . Assume one dimensional conduction through the sides, calculate the net heat flow rate and the inside surface temperature of the refrigerator. **[8]**

OR

- Q2)** a) State Fourier law of heat conduction and by using it derive an expression for steady state heat conduction through a long hollow cylinder of radii  $r_1$  and  $r_2$  maintains its two surfaces at temperatures,  $T_1$  and  $T_2$ , respectively. **[6]**
- b) Derive three dimensional generalize differential heat conduction equation in Cartesian coordinates and deduce it to one dimensional steady state heat conduction without heat generation. **[10]**

**P.T.O.**

- Q3) a)** Explain the concept of critical thickness of insulation on cylinder with the help of suitable illustration and sketches (s). **[6]**
- b)** A steam pipe of 5 cm inside diameter and 6.5 cm outside diameter is covered with a 2.75 cm radial thickness of high temperature insulation ( $k = 1.1 \text{ W/m.K}$ ). The surface heat transfer coefficient for inside and outside surfaces are  $4650 \text{ W/m}^2 \cdot \text{K}$  and  $11.5 \text{ W/m}^2 \cdot \text{K}$ , respectively. The thermal conductivity of the pipe material is  $45 \text{ W/m.K}$ . If the steam temperature is  $200^\circ\text{C}$  and ambient air temperature is  $25^\circ\text{C}$ , determine Heat loss per metre length of pipe. **[10]**

OR

- Q4) a)** Derive an expression for temperature distribution in a plane wall under steady state heat conduction with uniform heat generation. The wall is insulated on left surface and maintained at temperature  $T_s$  on right surface. **[8]**
- b)** A hollow sphere of inside radius 30 mm and outside radius 50 mm is electrically heated at its inner surface at a constant rate of  $105 \text{ W/m}^2$ . The outer surface is exposed to a fluid at  $30^\circ\text{C}$ , with heat transfer coefficient of  $170 \text{ W/m}^2 \cdot \text{K}$ . The thermal conductivity of the material is  $20 \text{ W/m.K}$ . Calculate inner and outer surface temperatures. **[8]**
- Q5) a)** Derive an expression for temperature distribution for unsteady state heat conduction using lumped heat capacity method. State assumptions in method. **[8]**
- b)** An aluminium sphere weighing 6 kg and initially at temperature of  $350^\circ\text{C}$  is suddenly immersed in a fluid at  $30^\circ\text{C}$  with convection coefficient of  $60 \text{ W/m}^2 \cdot \text{K}$ . Estimate the time required to cool the sphere to  $100^\circ\text{C}$ . Take thermophysical properties as  
 $C = 900 \text{ J/kg.K}$ ,  $\rho = 2700 \text{ kg/m}^3$ ,  $k = 205 \text{ W/m.K}$ . **[10]**

OR

- Q6) a)** What is the difference between fin effectiveness and fin efficiency? **[4]**
- b)** Explain the criteria of selection of fins. **[4]**
- c)** An aluminium alloy fin ( $k = 200 \text{ W/m.K}$ ), 3.5 mm thick and 2.5 cm long protrudes from a wall. The base is at  $420^\circ\text{C}$  and ambient air temperature is  $30^\circ\text{C}$ . The heat transfer coefficient may be taken as  $11 \text{ W/m}^2 \cdot \text{K}$ . Find the heat loss and fin efficiency, if the heat loss from fin tip is negligible. **[10]**

## SECTION - II

**Q7) a)** State: [8]

- i) Kirchoff's law of radiation,
- ii) Wien's displacement law,
- iii) Stefan Boltzman law,
- iv) Define diffuse body

b) A hot water radiator of overall dimensions  $2 \times 1 \times 0.2$  m is used to heat the room at  $18^\circ\text{C}$ . The surface temperature of radiator is  $60^\circ\text{C}$  and its surface is black. The actual surface of the radiator is 2.5 times the area of its envelop for convection for which the convection coefficient is given by  $h_c = 1.3 (\Delta T)^{1/3} \text{ W/m}^2.\text{K}$ . Calculate the rate of heat loss from the radiator by convection and radiation. [8]

OR

**Q8) a)** Define [4]

- i) White body, and
- ii) Opaque body.

b) Write a short note on gray body approximation. [4]

c) A black metal plate ( $k = 25 \text{ W/m.K}$ ) at  $300^\circ\text{C}$  is exposed to surrounding air at  $30^\circ\text{C}$ . It convects and radiates heat to surroundings. If the convection coefficient is  $25 \text{ W/m}^2.\text{K}$ . What is the temperature gradient in the plate? [8]

**Q9) a)** Explain the physical mechanism of convection heat transfer. [4]

b) Calculate the approximate Reynolds numbers and state if the flow is laminar or turbulent for a 10 m long yacht sailing at 13 km/h in sea water,  $\rho = 1000 \text{ kg/m}^3$  and  $\mu = 1.3 \times 10^{-3} \text{ kg/m.s}$ . [4]

c) Explain the Reynolds Colburn analogy for turbulent flow over a flat plate. [8]

OR

**Q10)a)** Discuss the dimensional analysis for forced convection heat transfer. **[8]**

- b) Air at 27° C is flowing across a tube with a velocity of 25 m/s. The tube could be either a square of 5 cm side or a circular cylinder of 5 cm dia. Compare the rate of heat transfer in each case, if the tube surface is at 127°C. **[8]**

Use the correlation:

$$Nu = C Re^n Pr^{1/3}$$

where,  $C = 0.027$ ,  $n = 0.805$  for cylinder

$C = 0.102$ ,  $n = 0.675$  for square tube.

Take the properties of air at 77°C = 350 K

$$\rho = 0.955 \text{ kg/m}^3, K_f = 0.03 \text{ W/m.K.}$$

$$\nu = 20.92 \times 10^{-6} \text{ m}^2/\text{s}, Pr = 0.7, \text{ and } C_p = 1.009 \text{ kJ/kg. K.}$$

**Q11)a)** What do you mean by fouling factor? State the causes of fouling? **[4]**

- b) Define effectiveness of heat exchanger. How is maximum heat transfer rate is obtained? **[4]**

- c) Steam enters a counter flow heat exchanger, dry saturated at 10 bar and 180°C. It leaves at 350°C. The mass flow rate of the steam is 720 kg/min. The hot gas enters the exchanger at 650°C with mass flow rate of 1320kg/min. If the tubes are 30 mm in diameter and 3 m long, determine the surface area of tubes required. Neglect the resistance offered by metallic tubes. Use following. **[10]**

data:

For steam  $C_{p.s.} = 2.71 \text{ kJ/kg. K}$ , and  $h_i = 600 \text{ W/m}^2.\text{K}$ . For gas  $C_{p.g} = 1 \text{ kJ/kg.K}$  and  $h_o = 250 \text{ W/m}^2.\text{K}$ .

OR

**Q12)a)** Compare film wise and dropwise condensation. **[4]**

b) A heat exchanger is required to cool 55,000 kg/h of alcohol from 66°C to 40°C in a parallel flow heat exchanger using 40,000 kg/h of water entering at 5°C. Calculate. **[10]**

i) Exit temperature of water,

ii) Heat transfer rate,

iii) Surface area required

Take overall heat transfer coefficient.

$$U = 580 \text{ W/m}^2.\text{K}$$

$$C_p (\text{alcohol}) = 3760 \text{ J/kg.K.}$$

$$C_p (\text{water}) = 4180 \text{ J/kg. K.}$$

c) State the limitations of LMTD methods. **[4]**

