Total No. of Questions – [3]

Total No. of Printed Pages: [2]

0		
R. No		
3.		

PAPER CODE UNG-243 (BSB)

May 2023 (ENDSEM) EXAM

S.Y. (Mechanical) (AY 2022-23 SEMESTER - II)

COURSE NAME: FLUID MECHANICS AND MACHINES

COURSE CODE: MEUA22203

(PATTERN 2020)

Time: [1Hr]

5)

[Max. Marks: 30]

(*) Instructions to candidates:

Use of scientific calculator is allowed

Use suitable data where ever required

All questions are compulsory

uestion o.	Question Description	Max.	со	BT
0.		Marks	mapped	Level
.1	a) Sketch Velocity and shear stress distribution for	[4]	[4]	[3]
	laminar flow through pipe. Also write mathematical			
	tion for them			
	to A relate of length 0.5 m and Width 0.25 m is placed	[6]	[4]	[3]
	in a fluid of specific gravity 0.9 and of			
	tic viscosity one stoke. If the fluid is moving with			
	legity of 5 m/s, then determine (i) friction drag on the			
	plate, (ii) thickness of boundary layer and (iii) shear			
	stress at the trailing edge of the plate. Use Blasius			
	solution			
	OR			
	c) Two parallel plates are 5 mm apart and a steady	[6]	[4]	[3]
	laminar flow of oil is occurring between them. If the			
	pressure drop is 10 kPa per meter length of the plates and			
	viscosity of oil is 0 06 Ns/m ² , then calculate (i) the			
	discharge per meter width, (ii) maximum shear stress and			
	(iii) maximum velocity of flow			
Q.2	a) Compare Francis and Kaplan turbine based on	[4]	[5]	[2]
	1) Head	[4]		[~]
	2) Type of flow,			
	3) Specific speed			
	4) Discharge			
	4) Discharge			- Amelia

b) A Pelton wheel working under a head of 70 m. It develops 100 kW shaft power when it runs at 220 rpm. Assume the speed ratio as 0.45, coefficient of velocity as 0.98 and overall efficiency as 85%. Determine i) Wheel diameter ii) Jet diameter iii) Jet ratio. OR c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flow ratio is 0.6, then determine (i) the diameter of runner, (ii) the diameter of	[4]
Assume the speed ratio as 0.45, coefficient of velocity as 0.98 and overall efficiency as 85%. Determine i) Wheel diameter ii) Jet diameter iii) Jet ratio. OR c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flav ratio is 0.6, then	[4]
c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flow rate of 6.6, then	[4]
c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flav ratio is 0.6, then	[4]
c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flaw rate of 0.6, then	[4]
c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flav ratio is 0.6, then	[4]
c) The hub diameter of a Kaplan turbine working under a head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flav ratio is 0.6, then	[4]
head of 15 m is 0.3 times the diameter of the runner. The turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flow rot; a is 0.6, then	
turbine is running at 90 rpm and the velocity of whirl at outlet is zero. If the vane angle of the extreme edge of the runner at outlet is 15° and the flavorer is 16° 0.6, then	
outlet is zero. If the vane angle of the extreme edge of the	
runner at over 1.5° and the Gray rate 3.6 0.6, then	
I fuller at outlet in 15° and the district a if V'	
determine (i) the diameter of runner (ii) the diameter of	
I DETERMINE IN The AMBRICA OF MANAGE IN THE	1
to the diameter of rullier, (11) to	
boss and (iii) discharge through the runner.	
1 effects of [4] [6]	[2]
a) Discuss 41 'testion phone and ellouis of	\ '
cavitation on the performance of nilmnd	
b) The external and internal diameters of the impeller of a [6]	[4]
b) The external and internal diameters of the	
centrifugal pump are 0.4 m and 0.2 m, respectively. The	
centrifical numering at 1700 rnm and its value is	
avit are set hook at an angle of 25°. If a consult tuding	
flow through the impeller is maintained at 2.5 ms, then	
determine (i) the inlet vane angle (ii) angle induce by	
absolute velocity at the outlet and (iii) work done by the	
impeller per unit weight of water.	
OR	
c) A three stage centrifugal pump delivers water at the	[4]
rate of 0.06 m ³ /s. Each impeller is 0.42 m in diameter and	
rate of 0.06 m ³ /s. Each impener is 0.42 m in change of the impellers is 0.024 m wide at the outlet. The speed of the impellers is	
950 rpm. The vanes are curved back at the outlet at an	ļ
angle of 45° and reduce the circumferential area by 10%.	
The overall efficiency is 78% and the manometric	
efficiency is 88%. Determine the head generated and the	
power consumed	

Note: [BT level - 1: Remember 2: Understand 3: Apply 4: Analyze 5: Evaluate 6: Create]